



# Build your own 2D simulator

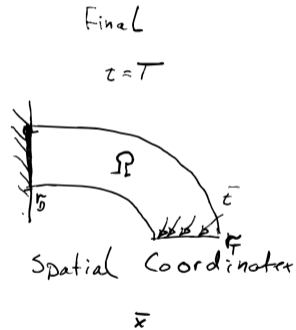
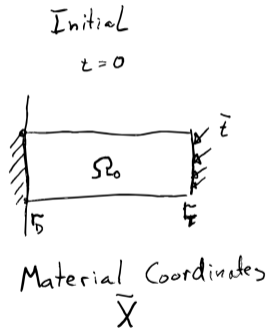
2D Hyperelastic materials using the Finite Volume Method

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# The Problem



## The Idealization Process (1/2)

We have the Cauchy equation (motion of equation).

$$\rho_0 \ddot{\mathbf{x}} = \mathbf{b}_0 + \nabla_{\mathbf{X}} \cdot \mathbf{P}, \quad \forall \mathbf{X} \in \Omega_0.$$

Subscript 0 means the physical quantity is given in material coordinate spaces,  $\mathbf{X}$ . Spatial coordinates are written as  $\mathbf{x}$  and not  $\mathbf{X}$ .

We will apply boundary conditions for a known traction field  $\mathbf{t}$  on the boundary,  $\Gamma_T$ ,

$$\mathbf{P}\mathbf{N} = \mathbf{t}, \quad \forall \mathbf{X} \in \Gamma_T$$

As well as Dirichlet conditions to keep a subset of the boundary fixed,  $\Gamma_D$ ,

$$\mathbf{x} = \text{const}, \quad \forall \mathbf{X} \in \Gamma_D$$

## The Idealization Process (2/2)

We use a constitutive equation between 2nd Piola–Kirchhoff stress tensor and Green strain tensor

$$\mathbf{S} = \lambda \operatorname{tr}(\mathbf{E}) \mathbf{I} + 2\mu \mathbf{E}$$

which is related to the 1st Piola Kirchhoff stress tensor

$$\mathbf{P} = \mathbf{F} \mathbf{S}$$

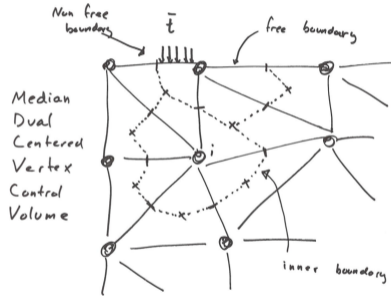
and the Green strain is by definition given as

$$\mathbf{E} = \frac{1}{2}(\mathbf{F}^T \mathbf{F} - \mathbf{I})$$

## Discretization Choices That is Given

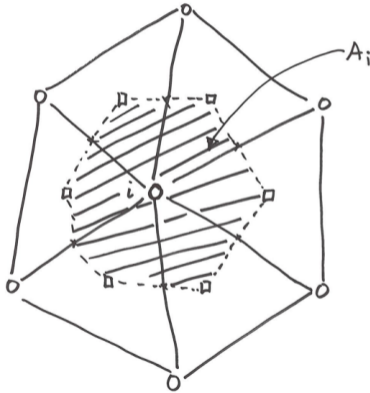
- We will use a finite volume method.
- We will use a 2D triangular computational domain.
- We will use median dual vertex centered control volume.
- We will assume that deformation gradients are constant over triangular elements.
- We will use simple first-order forward finite difference approximations in time.

## The Computational Mesh

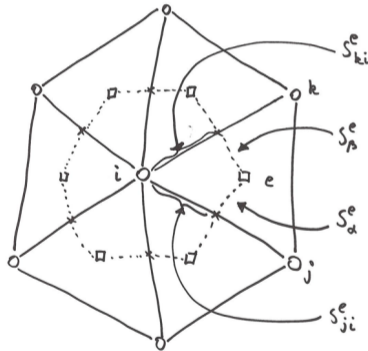


Observe we have three types of "real" boundaries,  $\Gamma_T$  and  $\Gamma_D$  are both non-free kinds of boundaries, and the third is the "free" boundary. The inner boundary is not a real boundary, it is just the interface between the computational cells.

## The Cell Notation



## The Boundary Notation



We use  $\gamma \in \{\alpha, \beta, ji, ki\}$  as index in summations so  $S_{\gamma}^e$  means  $S_{\alpha}^e$ ,  $S_{\beta}^e$ ,  $S_{ji}^e$ , and  $S_{ki}^e$ .



## Write up Volume Integrals

Let  $A_i$  denote the control volume of the  $i^{\text{th}}$  vertex

$$\int_{A_i} \rho_0 \ddot{\mathbf{x}}_i dA = \int_{A_i} \mathbf{b}_0 dA + \int_{A_i} \nabla_{\mathbf{x}} \cdot \mathbf{P} dA$$

With some hard work using Leibniz rule, Piecewise continuity of integrals, Midpoint approximation rule, and Gauss–Divergence Theorem. One derives

$$m_i \ddot{\mathbf{x}}_i = \underbrace{A_i \mathbf{b}_0}_{\equiv \mathbf{f}_i^{\text{ext}}} + \sum_e \sum_{\gamma} \int_{S_{\gamma}^e} \mathbf{P} \mathbf{N} dS$$

where  $\gamma$  runs over all boundary surfaces, ie. it can be  $\alpha, \beta$  or some  $ji, ki$ . The nodal mass is  $m_i = A_i \rho_0$ . The superscript on  $\mathbf{f}^{\text{ext}}$  indicates this force term is an external force on the system.

## The Boundary Conditions

If  $S_\gamma^e$  is on the boundary of the domain we can apply the boundary condition  $\mathbf{t} = \mathbf{PN}$ .

- On the “free” boundary we have  $\mathbf{t} = 0$  so

$$\int_{S_\gamma^e} \mathbf{PN} dS = 0$$

- For non-free we have prescribed known constant traction  $\mathbf{t}$  so

$$\int_{S_\gamma^e} \mathbf{PN} dS = \mathbf{t} l_\gamma^e$$

where  $l_\gamma^e$  is the length of piece wise linear boundary  $S_\gamma^e$ .

Meaning we can restrict the boundary summation to internal boundaries only.

## The Traction Forces

Thus,

- We split out the terms from non-free boundaries

$$m_i \ddot{\mathbf{x}}_i = \mathbf{f}_i^{\text{ext}} + \mathbf{f}_i^t + \sum_{\gamma} \int_{S_{\gamma}^e} \mathbf{P} \mathbf{N} dS$$

Now  $\gamma$  is only running over internal boundary edges ( $\alpha$  or  $\beta$  for each  $e$ ) and  $\delta$  over external boundaries

$$\mathbf{f}_i^t \equiv \sum_{\delta} \mathbf{t}_{\delta}^e$$

- The vector  $\mathbf{f}_i^t$  is termed the “nodal” traction force to distinguish it from the traction field  $\mathbf{t}$ .

## Computing the Deformation Gradient

Consider the  $e^{\text{th}}$  triangle consisting of nodes  $i$ ,  $j$  and  $k$ . Recall

$$d\mathbf{x} = \mathbf{F}d\mathbf{X}$$

Define  $\mathbf{g}_{ji}^e = \mathbf{x}_j - \mathbf{x}_i$  and  $\mathbf{G}_{ji}^e = \mathbf{X}_j - \mathbf{X}_i$  and so on. Now

$$\mathbf{g}_{ji}^e = \mathbf{F}^e \mathbf{G}_{ji}^e \text{ and } \mathbf{g}_{ki}^e = \mathbf{F}^e \mathbf{G}_{ki}^e$$

Putting it together

$$\underbrace{\begin{bmatrix} \mathbf{g}_{ji}^e & \mathbf{g}_{ki}^e \end{bmatrix}}_{\mathbf{D}^e} = \mathbf{F}^e \underbrace{\begin{bmatrix} \mathbf{G}_{ji}^e & \mathbf{G}_{ki}^e \end{bmatrix}}_{\mathbf{D}_0^e}$$

Now we can compute the deformation gradient as  $\mathbf{F}^e = \mathbf{D}^e (\mathbf{D}_0^e)^{-1}$ .

## Computing the Stress and Strain Tensors

As  $\mathbf{F}^e$  is known and constant over  $e$  then we may compute the Green strain tensor

$$\mathbf{E}^e = \frac{1}{2} \left( (\mathbf{F}^e)^T \mathbf{F}^e - \mathbf{I} \right)$$

and the 2nd Piola–Kirchhoff tensor

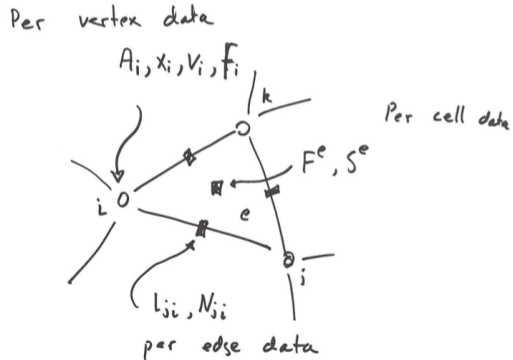
$$\mathbf{S}^e = \lambda \operatorname{tr}(\mathbf{E}^e) \mathbf{I} + 2\mu \mathbf{E}^e$$

and finally 1st Piola–Kirchhoff tensor

$$\mathbf{P}^e = \mathbf{F}^e \mathbf{S}^e$$

Observe that since  $\mathbf{F}^e$  is constant per triangle so will  $\mathbf{S}^e$  and  $\mathbf{P}^e$  be.

## The Grid Layout



Observe that  $\mathbf{P}^e$  and  $\mathbf{E}^e$  are also stored at face centers.

## Computing Boundary Integrals $\int_{S_\gamma^e} \mathbf{P} \mathbf{N} dS$

In the usual FVM approach one would apply the midpoint approximation rule, hence

$$\int_{S_\gamma^e} \mathbf{P} \mathbf{N} dS \approx \mathbf{P}^e \mathbf{N}_\gamma^e l_\gamma^e$$

Second thoughts

- $\mathbf{P}^e$  is constant over the entire triangle so we know its value at the midpoint
- We need to pre-compute  $\mathbf{N}_\gamma^e$  and  $l_\gamma^e$ .

As our control volume looks a bit “complex” the last parts can be a bit “difficult” to implement.

## The “Clever” Approach (1/2)

Observation: The closed surface integral over a constant tensor field is always zero

$$\oint_S \mathbf{P}^e \mathbf{N} dS = 0$$

For  $e^{\text{th}}$  triangle we make the “imaginary” closed surface integral

$$\int_{S_{ji}^e} \mathbf{P}^e \mathbf{N} dS + \int_{S_{jk}^e} \mathbf{P}^e \mathbf{N} dS + \int_{S_{\alpha}^e} \mathbf{P}^e \mathbf{N} dS + \int_{S_{\beta}^e} \mathbf{P}^e \mathbf{N} dS = 0$$

We then slightly rewrite this into

$$\int_{S_{\alpha}^e} \mathbf{P}^e \mathbf{N} dS + \int_{S_{\beta}^e} \mathbf{P}^e \mathbf{N} dS = - \int_{S_{ji}^e} \mathbf{P}^e \mathbf{N} dS - \int_{S_{jk}^e} \mathbf{P}^e \mathbf{N} dS$$



## The “Clever” Approach (2/2)

We have found the equivalence relation from the fact that we have constant stress tensors,

$$\int_{S_{\alpha}^e} \mathbf{P}^e \mathbf{N} dS + \int_{S_{\beta}^e} \mathbf{P}^e \mathbf{N} dS = - \int_{S_{ji}^e} \mathbf{P}^e \mathbf{N} dS - \int_{S_{jk}^e} \mathbf{P}^e \mathbf{N} dS$$

This means in computations we can now replace the sum of the surface integrals  $S_{\alpha}^e$  and  $S_{\beta}^e$  with the negative sum of the surface integrals  $S_{ji}^e$  and  $S_{jk}^e$ . The numerical values will be equivalent.

Having done the replacement we are now ready to apply the midpoint approximation rule

## The Midpoint Approximation Rule

After the replacement of the equivalent surface integrals we have,

$$\mathbf{f}_i^e \equiv - \int_{S_{ji}^e} \mathbf{P}^e \mathbf{N} dS - \int_{S_{jk}^e} \mathbf{P}^e \mathbf{N} dS$$

Using midpoint approximation rule and observing that  $\mathbf{l}_{ji} = \|\mathbf{X}_j - \mathbf{X}_i\|$ , we have,

$$\mathbf{f}_i^e \equiv -\frac{1}{2} \mathbf{P}^e \mathbf{N}_{ji}^e \mathbf{l}_{ji} - \frac{1}{2} \mathbf{P}^e \mathbf{N}_{ki}^e \mathbf{l}_{ki}$$

Thus, we have

$$m_i \ddot{\mathbf{X}}_i = \mathbf{f}_i^{\text{ext}} + \mathbf{f}_i^t + \sum_e \mathbf{f}_i^e$$

## Time Discretization

We have

$$m_i \ddot{\mathbf{x}}_i = \underbrace{\mathbf{f}_i^{\text{ext}} + \mathbf{f}_i^t + \sum_e \mathbf{f}_i^e}_{\equiv \mathbf{f}_i^{\text{total}}}$$

From nodal velocity  $\mathbf{v}_i = \dot{\mathbf{x}}_i$  we have coupled first order ODEs

$$\dot{\mathbf{v}}_i = \frac{1}{m_i} \mathbf{f}_i^{\text{total}}$$

$$\dot{\mathbf{x}}_i = \mathbf{v}_i$$

Apply first-order finite difference approximations

$$\mathbf{v}_i^{t+\Delta t} = \mathbf{v}_i^t + \frac{\Delta t}{m_i} \mathbf{f}_i^{\text{total}}$$

$$\mathbf{x}_i^{t+\Delta t} = \mathbf{x}_i^t + \Delta t \mathbf{v}_i^{t+\Delta t}$$

Notice that the velocity update is explicit whereas the position update is implicit.

## The Final Numerical Method

A single simulation step:

Step 1 Compute deformation gradients  $\mathbf{F}^e$  for all  $e$

Step 2 Compute Green strain tensors  $\mathbf{E}^e$  for all  $e$

Step 3 Compute 2nd Piola–Kirchhoff stress tensor  $\mathbf{S}^e$  for all  $e$

Step 4 Compute 1st Piola–Kirchhoff stress tensor  $\mathbf{P}^e$  for all  $e$

Step 5 Compute elastic forces  $\sum_e \mathbf{f}_i^e$  for all  $i$

Step 6 Compute total forces  $\mathbf{f}_i^{\text{total}}$  for all  $i$

Step 7 Compute velocity update  $\mathbf{v}_i^{t+\Delta t}$  for all  $i$

Step 8 Compute position update  $\mathbf{x}_i^{t+\Delta t}$  for all  $i$

Step 9 Increment time  $t \leftarrow t + \Delta t$

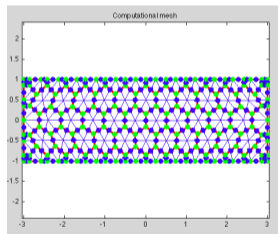
## The Input Parameters

- The triangle mesh
- The surface traction field  $\mathbf{t}$  (if any)
- The material density field  $\rho_0$
- The body force density field  $\mathbf{b}_0$
- The Lamé parameters  $\lambda$  and  $\mu$
- The time step size  $\Delta t$

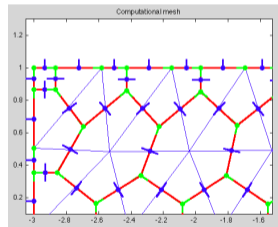
## Tip: Getting A Computational Mesh for Free

Code reusability is good:

- FVM handout has meshing tool for centroid dual vertex centered control volume



Computational Mesh



Zoomed View

This can be modified to a median dual vertex-centered control volume.

## The Model Collapse

Assume compression/extension in the  $x$ -axis then

$$\mathbf{x} = \begin{bmatrix} a & 0 \\ 0 & 1 \end{bmatrix} \mathbf{X}$$

$$\mathbf{F} = \frac{\partial \mathbf{x}}{\partial \mathbf{X}} = \begin{bmatrix} a & 0 \\ 0 & 1 \end{bmatrix}$$

$$\mathbf{C} = \mathbf{F}^T \mathbf{F} = \begin{bmatrix} a^2 & 0 \\ 0 & 1 \end{bmatrix}$$

$$\mathbf{E} = \frac{1}{2} (\mathbf{C} - \mathbf{I}) = \begin{bmatrix} \frac{1}{2} (a^2 - 1) & 0 \\ 0 & 0 \end{bmatrix}$$

$$\mathbf{S} = \lambda \text{tr}(\mathbf{E}) \mathbf{I} + 2\mu \mathbf{E} = \begin{bmatrix} \left(\mu + \frac{\lambda}{2}\right) (a^2 - 1) & 0 \\ 0 & \frac{\lambda}{2} (a^2 - 1) \end{bmatrix}$$

## The Model Collapse

Rewrite stress tensor

$$\begin{aligned}\mathbf{S} &= \begin{bmatrix} (\mu + \frac{\lambda}{2})(a^2 - 1) & 0 \\ 0 & \frac{\lambda}{2}(a^2 - 1) \end{bmatrix} \\ \mathbf{P} = \mathbf{FS} &= \begin{bmatrix} a & 0 \\ 0 & 1 \end{bmatrix} \begin{bmatrix} (\mu + \frac{\lambda}{2})(a^2 - 1) & 0 \\ 0 & \frac{\lambda}{2}(a^2 - 1) \end{bmatrix} \\ &= \begin{bmatrix} (\mu + \frac{\lambda}{2})(a^3 - a) & 0 \\ 0 & \frac{\lambda}{2}(a^2 - 1) \end{bmatrix}\end{aligned}$$



## The Model Collapse

Let  $\alpha = (\mu + \frac{\lambda}{2})$ ,  $\beta = \frac{\lambda}{2}$  then

$$\mathbf{P} = \begin{bmatrix} \alpha (a^3 - a) & 0 \\ 0 & \beta (a^2 - 1) \end{bmatrix}$$

We can consider  $\alpha > 0$  as Lamé first parameter  $\lambda$  is always positive and the second  $\mu$  is positive for most materials too.

## The Model Collapse

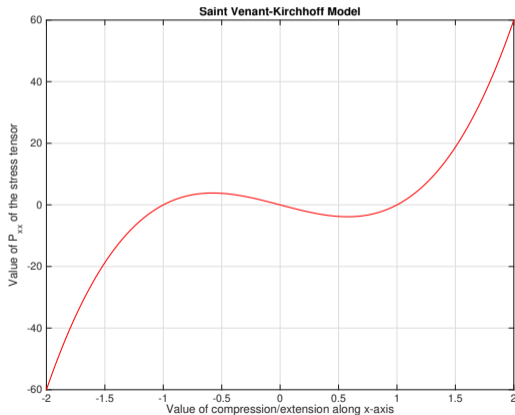
From

$$\mathbf{P}_{xx} = \alpha (a^3 - a)$$

We make the plot on the right.

Observe, no response for  $a = 0$  and  $a \pm 1$ .

For  $|a| < \frac{1}{\sqrt{3}} \approx 0.57$  material no longer resists compression.



## Assignment

- Show all the steps from the initial volume integral on top of Page 9 to the discrete form at the bottom that only contains the piece-wise linear surface integrals.
- Observe that we wrote the Cauchy equation using “material” space as the domain rather than the usual “spatial” setting given as

$$\rho \ddot{\mathbf{x}} = \mathbf{b} + \nabla \cdot \sigma$$

Note that in the material setting we have  $\mathbf{x}(\mathbf{X})$  whereas in the spatial setting we simply have  $\mathbf{x}$ . Show how to get from the spatial setting to the material setting<sup>1</sup>?

## Assignment

- Discuss computational benefits of  $\mathbf{F}^e = \mathbf{D}^e (\mathbf{D}_0^e)^{-1}$  on Page 12 Hint: What can be pre-computed?
- Discuss mesh properties needed to make sure  $\mathbf{D}_0^e$  is invertible.

## Assignment

- Assume a FEM approach with the local linear shape functions

$$w_i^e(\mathbf{X}) = \frac{A(\mathbf{X}_j, \mathbf{X}_k, \mathbf{X})}{A(\mathbf{X}_i, \mathbf{X}_j, \mathbf{X}_k)}$$

where the  $A$ 's are areas of respective triangles. For any point  $\mathbf{X}$  inside the triangle  $e$

$$\mathbf{x}(\mathbf{X}) \approx \sum_{a \in \{i, j, k\}} w_a^e(\mathbf{X}) \mathbf{x}_a$$

Compute  $\mathbf{F}(\mathbf{X}) = \frac{\partial \mathbf{x}(\mathbf{X})}{\partial \mathbf{X}}$

- Compare the FEM formula for  $\mathbf{F}$  against our formula from Page 12, what are the differences?

## Assignment

The constitutive equation on Page 13 needs material parameters  $\lambda$  and  $\mu$ .

- Find parameter values for the Lamé parameters  $\lambda$  and  $\mu$  for real known materials. Make a table of the values that you find.

**Remark:** It is beneficial to use cartoon-like parameter values at first when implementing and debugging a simulator. Stiff materials can make one fight ill-conditioned systems rather than finding the real bugs in the code. For instance, Young Modulus of 1000000.0 Pa, Poisson ratio of 0.3, and mass density of 1000  $\text{Kg}/\text{m}^3$ , and  $\Delta t = 0.001$  s.

## Assignment

Consider the theory covered from Page 15 to Page 18.

- Why is the formula

$$\mathbf{f}_i^e \equiv -\frac{1}{2} \mathbf{P}^e \mathbf{N}_{ji}^e \mathbf{l}_{ji} - \frac{1}{2} \mathbf{P}^e \mathbf{N}_{ki}^e \mathbf{l}_{ki}$$

more “convenient” than

$$\mathbf{f}_i^e \equiv \mathbf{P}^e \mathbf{N}_{\alpha}^e \mathbf{l}_{\alpha}^e + \mathbf{P}^e \mathbf{N}_{\beta}^e \mathbf{l}_{\beta}^e$$

Hint: consider whether you actually have to build the CV and whether the summation order has any importance.

## Assignment

- Analyze each input parameter on Page 21 – decide upon what its “value” should be and why
- Make hypotheses about how each input parameter affects a simulation
- Conduct experiments to verify your hypotheses

### Some Inspiration

- Examine the quality of triangle meshes
- Examine volume, momentum, mechanical energy conservation
- Examine stability depending on Lamé parameters and  $\Delta t$
- Examine symmetry behavior
- Examine robustness in relation to setting  $\mathbf{t}$
- ...



## Assignment

- From where was the value  $\frac{1}{\sqrt{3}}$  on Page 26 derived? Hint: Inflection points have slope zero.