



Strain and Stress Tensors

Introduction to Concept and
Definitions

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Modeling of Elastic Solids

The partial differential equation, Cauchy Equation, gives the equation of motion of an elastic solid

$$\rho \ddot{\mathbf{x}} = \mathbf{b} + \nabla \cdot \boldsymbol{\sigma}$$

The boundary conditions of the Cauchy stress tensor $\boldsymbol{\sigma}$ are given by

$$\boldsymbol{\sigma} \mathbf{n} = \mathbf{t}$$

where \mathbf{t} is a known applied surface traction. Finally, a constitutive law is needed that provides a relationship between stress and strain

$$\mathbf{S} = \frac{\partial \Psi}{\partial \mathbf{E}}$$

where \mathbf{S} is 2. Piola–Kirchhoff stress tensor and \mathbf{E} is the Green strain tensor. Now we will derive all these equations that we need.

Kinematics – Learning The Definitions of The Strain Tensors

The Material and Spatial Coordinates

We have two sets of coordinates

- Let \mathbf{X} be undeformed coordinates
- and \mathbf{x} the deformed coordinates

The deformation is given by the mapping from undeformed coordinates into deformed coordinates,

$$\mathbf{x} = \Phi(\mathbf{X}, t)$$

Observe

- The same global reference coordinate system is used for both sets of coordinates.
- Undeformed coordinates are sometimes also called material coordinates and deformed coordinates are the spatial coordinates.

The Deformation Gradient

By definition of differentials we have

$$d\mathbf{x} = \underbrace{\frac{\partial \Phi}{\partial \mathbf{X}}}_{\mathbf{F}} d\mathbf{X}$$

The partial derivative $\frac{\partial \Phi}{\partial \mathbf{X}} = \mathbf{F}$ is referred to as the Deformation gradient.

In particular, we have

$$\mathbf{F}_{ij} = \frac{\partial \Phi_i}{\partial \mathbf{X}_j} = \frac{\partial \mathbf{x}_i}{\partial \mathbf{X}_j}$$

Investigating Local Deformation Measures

Think of $d\mathbf{X}_1$ and $d\mathbf{X}_2$ as **arbitrary** chosen small “needles”. By definition we have

$$d\mathbf{x}_1 = \mathbf{F}d\mathbf{X}_1$$

$$d\mathbf{x}_2 = \mathbf{F}d\mathbf{X}_2$$

To describe any local deformation we investigate the dot products

$$d\mathbf{x}_1 \cdot d\mathbf{x}_2 \quad \text{and} \quad d\mathbf{X}_1 \cdot d\mathbf{X}_2$$

as these hold information about any local angle or length deformations.

The Right Cauchy–Green Deformation Tensor

First we will try to express $d\mathbf{x}_1 \cdot d\mathbf{x}_2$ in terms of material coordinates,

$$\begin{aligned}d\mathbf{x}_1 \cdot d\mathbf{x}_2 &= d\mathbf{x}_1^T d\mathbf{x}_2 \\ &= (\mathbf{F}d\mathbf{X}_1)^T (\mathbf{F}d\mathbf{X}_2) \\ &= d\mathbf{X}_1^T \underbrace{(\mathbf{F}^T \mathbf{F})}_{\mathbf{C}} d\mathbf{X}_2\end{aligned}$$

where

$$\mathbf{C} = \mathbf{F}^T \mathbf{F}$$

is called the right Cauchy–Green Deformation tensor.

Tensor Invariants (1/2)

The eigenvalues of \mathbf{C} are independent of rotation thus the coefficients of the characteristic polynomial are invariants

$$\det(\mathbf{C} - \mathbf{I}\lambda) = -\lambda^3 + I_C\lambda^2 - II_C^*\lambda + III_C = 0$$

where

$$I_C = \text{tr}(\mathbf{C}) = \mathbf{C} : \mathbf{I}$$

$$II_C^* = \frac{1}{2} \left(\text{tr}(\mathbf{C})^2 - \text{tr}(\mathbf{C}^2) \right)$$

$$III_C = \det(\mathbf{C})$$

Tensor Invariants (2/2)

We prefer an alternative definition of the second invariant

$$II_C = \text{tr}(\mathbf{C}^2) = \mathbf{C} : \mathbf{C}$$

as it simplifies later equations

The Eigenvalue Decomposition

Since \mathbf{C} is a symmetric positive definite we know that an eigenvalue decomposition exists

$$\mathbf{C}\mathbf{N} - \mathbf{N} = 0$$

where

$$\begin{aligned}\mathbf{N} &= [\mathbf{N}_1 \quad \mathbf{N}_2 \quad \mathbf{N}_3] \\ &= \begin{bmatrix} \lambda_1^2 & 0 & 0 \\ 0 & \lambda_2^2 & 0 \\ 0 & 0 & \lambda_3^2 \end{bmatrix}\end{aligned}$$

and $\mathbf{N}^T\mathbf{N} = \mathbf{N}\mathbf{N}^T = \mathbf{I}$ and $0 < \lambda_3^2 \leq \lambda_2^2 \leq \lambda_1^2 \in \mathbb{R}$. This can be rewritten as

$$\mathbf{C} = \mathbf{N}\mathbf{N}^T = \sum \lambda_i^2 (\mathbf{N}_i\mathbf{N}_i^T)$$

Invariants in terms of Eigenvalues

Observe that knowing the eigenvalues λ_i^2 of \mathbf{C} we can compute the tensor invariants as

$$I_C = \lambda_1^2 + \lambda_2^2 + \lambda_3^2$$

$$II_C = \lambda_1^4 + \lambda_2^4 + \lambda_3^4$$

$$III_C = \lambda_1^2 \lambda_2^2 \lambda_3^2$$

These formulas will become handy when we later wish to change coordinates.

A Rotation Tensor

The square root of \mathbf{C} is

$$\mathbf{C} = \sqrt{\mathbf{C}}\sqrt{\mathbf{C}} = \mathbf{U}\mathbf{U}$$

with

$$\mathbf{U} = \sum_i \lambda_i (\mathbf{N}_i \mathbf{N}_i^T)$$

Let us define the tensor \mathbf{R} as

$$\mathbf{R} = \mathbf{F}\mathbf{U}^{-1}$$

We can then prove that \mathbf{R} is a rotation tensor

$$\mathbf{R}^T \mathbf{R} = \mathbf{U}^{-T} \mathbf{F}^T \mathbf{F} \mathbf{U}^{-1} = \mathbf{U}^{-T} \mathbf{C} \mathbf{U}^{-1} = \mathbf{U}^{-T} \mathbf{U} \mathbf{U} \mathbf{U}^{-1} = \mathbf{I}$$

The Polar Decomposition of Deformation Gradient

The new rotation tensor \mathbf{R} implies that \mathbf{F} can be decomposed as

$$\mathbf{F} = \mathbf{R}\mathbf{U}$$

The implication is

$$d\mathbf{x} = \mathbf{F}d\mathbf{X} = \mathbf{R}(\mathbf{U}d\mathbf{X})$$

where $(\mathbf{U}d\mathbf{X})$ is a stretch along the eigenvectors and \mathbf{R} is a rotation of the stretched vector.

Therefore

- \mathbf{U} is called the stretch tensor
- \mathbf{R} is called the rotation tensor

The Left Cauchy–Green Deformation Tensor

Next we will rewrite $d\mathbf{X}_1 \cdot d\mathbf{X}_2$ in terms of spatial coordinates,

$$\begin{aligned}d\mathbf{X}_1 \cdot d\mathbf{X}_2 &= d\mathbf{X}_1^T d\mathbf{X}_2 \\ &= (\mathbf{F}^{-1}d\mathbf{x}_1)^T (\mathbf{F}^{-1}d\mathbf{x}_2) \\ &= d\mathbf{x}_1^T \underbrace{\left(\mathbf{F}\mathbf{F}^T \right)}_{\mathbf{B}}^{-1} d\mathbf{x}_2\end{aligned}$$

where

$$\mathbf{B} = \mathbf{F}\mathbf{F}^T$$

is called the left Cauchy–Green Deformation tensor.

The Lagrange–Green Strain Tensor

We look at the deformation change. The difference between spatial and material dot products expressed in material coordinates,

$$\begin{aligned}\frac{1}{2} (d\mathbf{x}_1 \cdot d\mathbf{x}_2 - d\mathbf{X}_1 \cdot d\mathbf{X}_2) &= \frac{1}{2} (d\mathbf{x}_1^T d\mathbf{x}_2 - d\mathbf{X}_1^T d\mathbf{X}_2) \\ &= \frac{1}{2} \left((\mathbf{F}d\mathbf{X}_1)^T (\mathbf{F}d\mathbf{X}_2) - d\mathbf{X}_1^T d\mathbf{X}_2 \right) \\ &= d\mathbf{X}_1^T \underbrace{\frac{1}{2} (\mathbf{F}^T \mathbf{F} - \mathbf{I})}_{\mathbf{E}} d\mathbf{X}_2\end{aligned}$$

where

$$\mathbf{E} = \frac{1}{2} (\mathbf{F}^T \mathbf{F} - \mathbf{I}) = \frac{1}{2} (\mathbf{C} - \mathbf{I})$$

is called the Green Strain tensor.

The Euler–Almansi Strain Tensor

We describe the deformation change in terms of spatial coordinates,

$$\begin{aligned}\frac{1}{2} (d\mathbf{x}_1 \cdot d\mathbf{x}_2 - d\mathbf{X}_1 \cdot d\mathbf{X}_2) &= \frac{1}{2} (d\mathbf{x}_1^T d\mathbf{x}_2 - d\mathbf{X}_1^T d\mathbf{X}_2) \\ &= \frac{1}{2} \left(d\mathbf{x}_1^T d\mathbf{x}_2 - (\mathbf{F}^{-1} d\mathbf{x}_1)^T (\mathbf{F}^{-1} d\mathbf{x}_2) \right) \\ &= d\mathbf{x}_1^T \underbrace{\frac{1}{2} \left(\mathbf{I} - (\mathbf{F}\mathbf{F}^T)^{-1} \right)}_{\mathbf{e}} d\mathbf{x}_2\end{aligned}$$

where

$$\mathbf{e} = \frac{1}{2} \left(\mathbf{I} - (\mathbf{F}\mathbf{F}^T)^{-1} \right) = \frac{1}{2} \left(\mathbf{I} - \mathbf{B}^{-1} \right)$$

is called the Euler Strain tensor.

Spatial to Material Tensor Conversions

Observe that

$$\mathbf{e} = \mathbf{F}^{-T} \mathbf{E} \mathbf{F}^{-1}$$

$$\mathbf{E} = \mathbf{F}^T \mathbf{e} \mathbf{F}$$

These are general tensor conversion formulas.

The Displacement Field

Let us define the displacement field as follows

$$\mathbf{u} = \mathbf{x} - \mathbf{X}$$

Then we trivially have

$$\mathbf{x} = \mathbf{u} + \mathbf{X}$$

and

$$\mathbf{F} = \frac{\partial(\mathbf{u} + \mathbf{X})}{\partial \mathbf{X}} = \frac{\partial \mathbf{u}}{\partial \mathbf{X}} + \mathbf{I}$$

or

$$\frac{\partial \mathbf{u}}{\partial \mathbf{X}} = \mathbf{F} - \mathbf{I}$$

Putting Displacement Field into Play

Let us use $\mathbf{F} = \frac{\partial \mathbf{u}}{\partial \mathbf{X}} + \mathbf{I}$ in the definition of the Green Strain Tensor

$$\begin{aligned}\mathbf{E} &= \frac{1}{2} \left(\left(\frac{\partial \mathbf{u}}{\partial \mathbf{X}} + \mathbf{I} \right)^T \left(\frac{\partial \mathbf{u}}{\partial \mathbf{X}} + \mathbf{I} \right) - \mathbf{I} \right) \\ &= \frac{1}{2} \left(\left(\frac{\partial \mathbf{u}^T}{\partial \mathbf{X}} \frac{\partial \mathbf{u}}{\partial \mathbf{X}} \right) + \left(\mathbf{I}^T \frac{\partial \mathbf{u}}{\partial \mathbf{X}} \right) + \left(\frac{\partial \mathbf{u}^T}{\partial \mathbf{X}} \mathbf{I} \right) + (\mathbf{I}^T \mathbf{I}) - \mathbf{I} \right) \\ &= \frac{1}{2} \left(\frac{\partial \mathbf{u}^T}{\partial \mathbf{X}} \frac{\partial \mathbf{u}}{\partial \mathbf{X}} + \frac{\partial \mathbf{u}}{\partial \mathbf{X}} + \frac{\partial \mathbf{u}^T}{\partial \mathbf{X}} \right)\end{aligned}$$

Assuming Small Displacement Gradients

If $\| \frac{\partial \mathbf{u}}{\partial \mathbf{X}} \| \ll 1$ then $\frac{\partial \mathbf{u}}{\partial \mathbf{X}}^T \frac{\partial \mathbf{u}}{\partial \mathbf{X}} \approx \mathbf{0}$ and we have

$$\mathbf{E} \approx \varepsilon_0 = \frac{1}{2} \left(\frac{\partial \mathbf{u}}{\partial \mathbf{X}}^T + \frac{\partial \mathbf{u}}{\partial \mathbf{X}} \right)$$

This strain tensor is called the Cauchy Strain tensor. If we used \mathbf{e} instead of \mathbf{E} we would have found

$$\mathbf{e} \approx \varepsilon = \frac{1}{2} \left(\frac{\partial \mathbf{u}}{\partial \mathbf{x}}^T + \frac{\partial \mathbf{u}}{\partial \mathbf{x}} \right)$$

However, for small displacement gradients $\mathbf{x} \approx \mathbf{X}$, $\frac{\partial \mathbf{u}}{\partial \mathbf{x}} \approx \frac{\partial \mathbf{u}}{\partial \mathbf{X}}$, and so $\varepsilon_0 \approx \varepsilon$.

The Strain Tensors

Green Strain Tensor

$$\mathbf{E} = \frac{1}{2} \left(\frac{\partial \mathbf{u}}{\partial \mathbf{X}}^T \frac{\partial \mathbf{u}}{\partial \mathbf{X}} + \frac{\partial \mathbf{u}}{\partial \mathbf{X}} + \frac{\partial \mathbf{u}}{\partial \mathbf{X}}^T \right)$$

Cauchy Strain Tensor

$$\boldsymbol{\varepsilon} \approx \boldsymbol{\varepsilon}_0 = \frac{1}{2} \left(\frac{\partial \mathbf{u}}{\partial \mathbf{X}}^T + \frac{\partial \mathbf{u}}{\partial \mathbf{X}} \right)$$

Observe that using $\frac{\partial \mathbf{u}}{\partial \mathbf{X}} = \mathbf{F} - \mathbf{I}$ we have

$$\boldsymbol{\varepsilon}_0 = \frac{1}{2} (\mathbf{F}^T + \mathbf{F}) - \mathbf{I}$$

The Velocity

Since $\mathbf{x} = \Phi(\mathbf{X}, t)$ then the spatial velocity of a material point is

$$\mathbf{v}(\mathbf{X}, t) \equiv \dot{\mathbf{x}}(\mathbf{X}, t) = \frac{\partial \mathbf{x}(\mathbf{X}, t)}{\partial t} = \frac{\partial \Phi(\mathbf{X}, t)}{\partial t}$$

or given in terms of spatial coordinates

$$\mathbf{v}(\mathbf{x}, t) = \mathbf{v}(\Phi^{-1}(\mathbf{x}, t), t)$$

The Velocity Gradient

The velocity gradient wrt. spatial coordinates is then

$$\mathbf{V} \equiv \frac{\partial \mathbf{v}(\mathbf{x}, t)}{\partial \mathbf{x}}$$

Observe (Notice $\frac{\partial \mathbf{X}}{\partial t} = 0$)

$$\dot{\mathbf{F}} = \frac{d}{dt} \left(\frac{\partial \Phi}{\partial \mathbf{X}} \right) = \frac{\partial}{\partial \mathbf{X}} \left(\frac{\partial \Phi}{\partial t} \right) = \frac{\partial}{\partial \mathbf{X}} \mathbf{v} = \frac{\partial \mathbf{v}}{\partial \mathbf{x}} \frac{\partial \Phi}{\partial \mathbf{X}} = \mathbf{V} \mathbf{F}$$

Resulting in

$$\mathbf{V} = \dot{\mathbf{F}} \mathbf{F}^{-1}$$

The Rate of Deformation

Taking the derivative

$$\frac{d}{dt} (d\mathbf{x}_1 \cdot d\mathbf{x}_2) = d\mathbf{X}_1^T \dot{\mathbf{C}} d\mathbf{X}_2 = 2d\mathbf{X}_1^T \dot{\mathbf{E}} d\mathbf{X}_2$$

because $\mathbf{E} = \frac{1}{2}(\mathbf{C} - \mathbf{I})$. The material strain rate tensor is

$$\dot{\mathbf{E}} = \frac{1}{2}\dot{\mathbf{C}} = \frac{1}{2}(\dot{\mathbf{F}}^T \mathbf{F} + \mathbf{F}^T \dot{\mathbf{F}})$$

Using $d\mathbf{X} = \mathbf{F}^{-1}d\mathbf{x}$ we have

$$\frac{1}{2} \frac{d}{dt} (d\mathbf{x}_1 \cdot d\mathbf{x}_2) = d\mathbf{x}_1^T \underbrace{\mathbf{F}^{-T} \dot{\mathbf{E}} \mathbf{F}^{-1}}_{\mathbf{D}} d\mathbf{x}_2$$

where \mathbf{D} is the rate of deformation tensor

Coordinate Conversion of Rate Tensors

Observe that similar to the Euler–Almansi and Lagrange–Green strain tensors we have

$$\begin{aligned}\mathbf{D} &= \mathbf{F}^{-T} \dot{\mathbf{E}} \mathbf{F}^{-1} \\ \dot{\mathbf{E}} &= \mathbf{F}^T \mathbf{D} \mathbf{F}\end{aligned}$$

From this, we also find that

$$\begin{aligned}\mathbf{D} &= \mathbf{F}^{-T} \dot{\mathbf{E}} \mathbf{F}^{-1} \\ &= \mathbf{F}^{-T} \frac{1}{2} \left(\dot{\mathbf{F}}^T \mathbf{F} + \mathbf{F}^T \dot{\mathbf{F}} \right) \mathbf{F}^{-1} \\ &= \frac{1}{2} (\mathbf{V} + \mathbf{V}^T)\end{aligned}$$

because $\mathbf{V} = \dot{\mathbf{F}} \mathbf{F}^{-1}$. Thus \mathbf{D} is the symmetric part of the velocity gradient.

Dynamics – Learning The Definitions of The Stress Tensors and Power

A Spatial Surface Element

The spatial surface element $d\mathbf{s}$ can be written as

$$d\mathbf{s} = \begin{bmatrix} ds_1 \\ ds_2 \\ ds_3 \end{bmatrix}$$

where ds_i is the projection of the area of surface $d\mathbf{s}$ onto the i^{th} coordinate axis.

Observe that the surface normal is given by

$$\mathbf{n} = \frac{d\mathbf{s}}{\|d\mathbf{s}\|}$$

Cauchy's Stress Hypothesis

Definition: Stress is force per unit area. If we apply the force $d\mathbf{f}$ to the surface element $d\mathbf{s}$

$$d\mathbf{f}_1 = \sigma_{11}d\mathbf{s}_1 + \sigma_{12}d\mathbf{s}_2 + \sigma_{13}d\mathbf{s}_3$$

$$d\mathbf{f}_2 = \sigma_{21}d\mathbf{s}_1 + \sigma_{22}d\mathbf{s}_2 + \sigma_{23}d\mathbf{s}_3$$

$$d\mathbf{f}_3 = \sigma_{31}d\mathbf{s}_1 + \sigma_{32}d\mathbf{s}_2 + \sigma_{33}d\mathbf{s}_3$$

where σ_{ij} depends on position and time, collecting them

$$\sigma = [\sigma_{ij}]$$

we have

$$d\mathbf{f} = \sigma d\mathbf{s}$$

where σ is the Cauchy Stress Tensor.

The First Piola–Kirchhoff Stress Tensor

By definition the Cauchy stress tensor

$$d\mathbf{f} = \boldsymbol{\sigma} d\mathbf{s}$$

The first Piola–Kirchhoff stress tensor is defined as

$$d\mathbf{f} = \mathbf{P} d\mathbf{S}$$

So we must have

$$\boldsymbol{\sigma} d\mathbf{s} = d\mathbf{f} = \mathbf{P} d\mathbf{S}$$

Using Nanson's Relation

$$\sigma_j \mathbf{F}^{-T} d\mathbf{S} = d\mathbf{f} = \mathbf{P} d\mathbf{S}$$

From this we have the relation between the Cauchy stress tensor and the first Piola–Kirchhoff stress tensor

The Second Piola–Kirchhoff Stress Tensor

By definition the Cauchy stress tensor

$$d\mathbf{f} = \boldsymbol{\sigma} d\mathbf{s}$$

The second Piola–Kirchhoff stress tensor is defined as

$$d\mathbf{F} = \mathbf{S} d\mathbf{S}$$

However $d\mathbf{f} = \mathbf{F} d\mathbf{F}$ so we must have

$$\boldsymbol{\sigma} d\mathbf{s} = d\mathbf{f} = \mathbf{F} d\mathbf{F} = \mathbf{F} \mathbf{S} d\mathbf{S}$$

$$\boldsymbol{\sigma} d\mathbf{s} = \mathbf{F} \mathbf{S} d\mathbf{S}$$

$$j\mathbf{F}^{-1}\boldsymbol{\sigma}\mathbf{F}^{-T} d\mathbf{S} = \mathbf{S} d\mathbf{S}$$

So

The Traction Force

We have

$$d\mathbf{f} = \sigma d\mathbf{s}$$

If we divide by surface area we will have traction $\mathbf{t} = d\mathbf{f} / \|d\mathbf{s}\|$

$$\mathbf{t} = \sigma \mathbf{n}$$

where

$$\begin{bmatrix} \mathbf{t}_1 \\ \mathbf{t}_2 \\ \mathbf{t}_3 \end{bmatrix} = \begin{bmatrix} \sigma_{11} & \sigma_{12} & \sigma_{13} \\ \sigma_{21} & \sigma_{22} & \sigma_{23} \\ \sigma_{31} & \sigma_{32} & \sigma_{33} \end{bmatrix} \begin{bmatrix} \mathbf{n}_1 \\ \mathbf{n}_2 \\ \mathbf{n}_3 \end{bmatrix}$$

Integration of Forces

Given the body volume force density \mathbf{b} and arbitrary spatial volume element v with spatial surface s then the total force on the volume is found by intergration

$$\begin{aligned}\mathcal{F} &= \int_v \mathbf{b} dv + \oint_s \mathbf{t} ds \\ &= \int_v \mathbf{b} dv + \oint_s (\boldsymbol{\sigma} \mathbf{n}) ds\end{aligned}$$

Using Volume Integrals

Recall Gauss–Divergence Theorem, for tensor field \mathbf{A} over any closed volume v with surface s

$$\int_v \nabla \cdot \mathbf{A} dv = \oint_s \mathbf{A} \mathbf{n} ds$$

Applying this to

$$\mathcal{F} = \int_v \mathbf{b} dv + \oint_s \sigma \mathbf{n} ds$$

yields

$$\mathcal{F} = \int_v \underbrace{(\mathbf{b} + \nabla \cdot \sigma)}_{\mathbf{f}^*} dv$$

The Total Force and the Effective Force

In mechanical equilibrium, the effective force \mathbf{f}^* must be zero since v was chosen arbitrarily. Thus writing it out

$$\mathbf{b} + \nabla \cdot \sigma = 0$$

This is Cauchy's Equation of Equilibrium. In non equilibrium, we must take inertia forces into account, $\mathcal{F} = \int_v \rho \ddot{\mathbf{x}} dv$,

$$\rho \ddot{\mathbf{x}} = \mathbf{b} + \nabla \cdot \sigma$$

This is the motion of the equation in spatial coordinates for any continuum.

Recap Work and Power Definitions

Let $\mathbf{x}(t)$ be a particle and \mathbf{f} some force acting on the particle. Work is force times distance, for a constant force over a displacement \mathbf{u}

$$W = \mathbf{f} \cdot \mathbf{u}$$

Or more general

$$W = \int \mathbf{f} \cdot d\mathbf{x}$$

Power is the rate at which work is performed

$$P \equiv \frac{dW}{dt}$$

The instantaneous power is $P = \mathbf{f} \cdot \mathbf{v}$ and so $W = \int_0^t \mathbf{f} \cdot \mathbf{v} dt$.

Power of The Effective Force

The power from the effective force

$$p = \mathbf{f}^* \cdot \mathbf{v}$$

The total power applied by the effective force

$$P = \int_V p dv = \int_V \mathbf{f}^* \cdot \mathbf{v} dv$$

Rewriting into

$$\begin{aligned} P &= \int_V (\mathbf{b} + (\nabla \cdot \sigma)) \cdot \mathbf{v} dv \\ &= \int_V (\nabla \cdot \sigma) \cdot \mathbf{v} dv + \int_V \mathbf{b} \cdot \mathbf{v} dv \end{aligned}$$

Using the “Product” Rule

Using

$$\nabla \cdot (\boldsymbol{\sigma} \mathbf{v}) = (\nabla \cdot \boldsymbol{\sigma}) \cdot \mathbf{v} + \boldsymbol{\sigma} : (\nabla \mathbf{v}^T)$$

Then

$$P = \int_{\mathcal{V}} (\nabla \cdot \boldsymbol{\sigma}) \cdot \mathbf{v} dv + \int_{\mathcal{V}} \mathbf{b} \cdot \mathbf{v} dv$$

Becomes

$$P = \int_{\mathcal{V}} \nabla \cdot (\boldsymbol{\sigma} \mathbf{v}) dv - \int_{\mathcal{V}} \boldsymbol{\sigma} : \mathbf{V}^T dv + \int_{\mathcal{V}} \mathbf{b} \cdot \mathbf{v} dv$$

Recall $\nabla \mathbf{v}^T = \mathbf{V}^T$.

Apply Gauss–Divergence Theorem

Using Gauss–Divergence Theorem

$$P = \oint_S \mathbf{n} \cdot (\boldsymbol{\sigma} \mathbf{v}) ds - \int_V \boldsymbol{\sigma} : \mathbf{V}^T dv + \int_V \mathbf{b} \cdot \mathbf{v} dv$$

Using $\mathbf{t} = \boldsymbol{\sigma} \mathbf{n}$ and $\boldsymbol{\sigma}^T = \boldsymbol{\sigma}$

$$P = \oint_S \mathbf{t} \cdot \mathbf{v} ds - \int_V \boldsymbol{\sigma} : \mathbf{V} dv + \int_V \mathbf{b} \cdot \mathbf{v} dv$$

Using $\mathbf{D} = \frac{1}{2} (\mathbf{V} + \mathbf{V}^T)$ and symmetry of $\boldsymbol{\sigma}$

$$P = \oint_S \mathbf{t} \cdot \mathbf{v} ds - \int_V \boldsymbol{\sigma} : \mathbf{D} dv + \int_V \mathbf{b} \cdot \mathbf{v} dv$$

The Power in Spatial Coordinates

The internal stress power term in spatial coordinates

$$P_e = \int_{\mathcal{V}} \boldsymbol{\sigma} : \mathbf{V} dv = \int_{\mathcal{V}} \boldsymbol{\sigma} : \mathbf{D} dv$$

Thus, in terms of power, we say that $\boldsymbol{\sigma}$ is work conjugate to \mathbf{D} . Our next task is to rewrite the internal power in terms of material coordinates. The result is to find the work conjugate quantities of

- The First Piola–Kirchhoff Stress tensor
- The Second Piola–Kirchhoff Stress tensor

Power Conjugacy of \mathbf{P}

$$P_e = \int_v \boldsymbol{\sigma} : \mathbf{V} dv \quad (\text{by definition})$$

$$= \int_V j\boldsymbol{\sigma} : \mathbf{V} dV \quad (\text{by } dv = j dV)$$

$$= \int_V j\boldsymbol{\sigma} : (\dot{\mathbf{F}}\mathbf{F}^{-1}) dV \quad (\text{by } \mathbf{V} = \dot{\mathbf{F}}\mathbf{F}^{-1})$$

$$= \int_V \text{tr} \left(j\boldsymbol{\sigma} (\dot{\mathbf{F}}\mathbf{F}^{-1}) \right) dV \quad (\text{by } \mathbf{A} : \mathbf{B} = \text{tr} (\mathbf{A}^T \mathbf{B}))$$

$$= \int_V \text{tr} \left((j\mathbf{F}^{-1}\boldsymbol{\sigma}) \dot{\mathbf{F}} \right) dV \quad (\text{tr} (\mathbf{AB}) = \text{tr} (\mathbf{BA}))$$

$$= \int_V \mathbf{P} : \dot{\mathbf{F}} dV \quad (\text{by } \text{tr} (\mathbf{A}^T \mathbf{B}) = \mathbf{A} : \mathbf{B})$$

Power Conjugacy of \mathbf{S}

$$P_e = \int_v \boldsymbol{\sigma} : \mathbf{D} dv \quad (\text{by definition})$$

$$= \int_V j \boldsymbol{\sigma} : \mathbf{D} dV \quad (\text{by } dv = j dV)$$

$$= \int_V j \boldsymbol{\sigma} : (\mathbf{F}^{-T} \dot{\mathbf{E}} \mathbf{F}^{-1}) dV \quad (\text{by } \mathbf{D} = \mathbf{F}^{-T} \dot{\mathbf{E}} \mathbf{F}^{-1})$$

$$= \int_V \text{tr} \left(j \boldsymbol{\sigma} (\mathbf{F}^{-T} \dot{\mathbf{E}} \mathbf{F}^{-1}) \right) dV \quad (\text{by } \mathbf{A} : \mathbf{B} = \text{tr} (\mathbf{A}^T \mathbf{B}))$$

$$= \int_V \text{tr} \left((j \mathbf{F}^{-1} \boldsymbol{\sigma} \mathbf{F}^{-T}) \dot{\mathbf{E}} \right) dV \quad (\text{by } \text{tr} (\mathbf{A} \mathbf{B}) = \text{tr} (\mathbf{B} \mathbf{A}))$$

$$= \int_V \mathbf{S} : \dot{\mathbf{E}} dV \quad (\text{by } \text{tr} (\mathbf{A}^T \mathbf{B}) = \mathbf{A} : \mathbf{B})$$

The Energy Balance

For the equilibrium system mechanical energy balance means that $P = 0$. That is the time derivative of mechanical energy is zero – constant in time.

In the case of a non-equilibrium system we must include inertia forces,

$$P = \int_V (\rho \ddot{\mathbf{x}} - \mathbf{b} - \nabla \cdot \boldsymbol{\sigma}) \cdot \mathbf{v} dv$$

Repeating the previous derivations to the total power becomes

$$P = \int_V \rho \ddot{\mathbf{x}} \cdot \mathbf{v} dv + \int_V \boldsymbol{\sigma} : \mathbf{D} dv - \oint_S \mathbf{t} \cdot \mathbf{v} dv - \int_V \mathbf{b} \cdot \mathbf{v} dv$$

and it must be zero for the conservation of mechanical energy.

Constitutive Equations

The Strain Energy in terms of \mathbf{P}

The total strain energy Ψ (The time integral of the “elastic” power P_e),

$$\Psi(\mathbf{F}) = \int_0^t \underbrace{\mathbf{P} : \dot{\mathbf{F}}}_{=\dot{\Psi}} dt$$

By the chain rule, we have

$$\dot{\Psi} = \frac{d}{dt} \Psi(\mathbf{F}) = \frac{\partial \Psi}{\partial \mathbf{F}} : \dot{\mathbf{F}}$$

So

$$\mathbf{P} : \dot{\mathbf{F}} = \dot{\Psi} = \frac{\partial \Psi}{\partial \mathbf{F}} : \dot{\mathbf{F}}$$

and we must have

$$\mathbf{P} = \frac{\partial \Psi}{\partial \mathbf{F}}$$

The Strain Energy in terms of \mathbf{S}

The total strain energy Ψ ,

$$\Psi(\mathbf{F}) = \int_0^t \underbrace{\mathbf{S} : \dot{\mathbf{E}}}_{=\dot{\Psi}} dt$$

By the chain rule, we have

$$\dot{\Psi} = \frac{d}{dt} \Psi(\mathbf{E}) = \frac{\partial \Psi}{\partial \mathbf{E}} : \dot{\mathbf{E}}$$

So

$$\mathbf{S} : \dot{\mathbf{E}} = \dot{\Psi} = \frac{\partial \Psi}{\partial \mathbf{E}} : \dot{\mathbf{E}}$$

and we must have

$$\mathbf{S} = \frac{\partial \Psi}{\partial \mathbf{E}}$$

Isotropic Materials

The potential function is a function of invariants of \mathbf{C}

$$\Psi(\mathbf{C}) = \Psi(I_C, II_C, III_C)$$

The Second Piola–Kirchhoff Stress Tensor computed as

$$\mathbf{S} = 2 \frac{\partial \Psi}{\partial I_C} \mathbf{I} + 4 \frac{\partial \Psi}{\partial II_C} \mathbf{C} + 2j^2 \frac{\partial \Psi}{\partial III_C} \mathbf{C}^{-1}$$

The Saint Venant-Kirchhoff (SVK) Material Model

The simplest hyperelastic material model. The strain-energy is

$$\Psi(\mathbf{E}) = \frac{\lambda}{2} (\text{tr}(\mathbf{E}))^2 + \mu \text{tr}(\mathbf{E}^2)$$

where λ and μ are the Lamé constants.

From $\mathbf{S} = \frac{\partial \Psi}{\partial \mathbf{E}}$ we have the stress-strain relation

$$\mathbf{S} = \lambda \text{tr}(\mathbf{E}) \mathbf{I} + 2\mu \mathbf{E}$$

The **S-C** Relation using the SVK-Material

By definition $2\mathbf{E} + \mathbf{I} = \mathbf{C}$

$$\mathbf{S} = \frac{\partial \Psi(\mathbf{C})}{\partial \mathbf{C}} \frac{\partial \mathbf{C}}{\partial \mathbf{E}} = 2 \frac{\partial \Psi}{\partial \mathbf{C}}$$

For isotropic materials stretch is independent of rotation $\Psi(\mathbf{C}) = \Psi(I_C, II_C, III_C)$

$$\mathbf{S} = 2 \left(\frac{\partial \Psi}{\partial I_C} \frac{\partial I_C}{\partial \mathbf{C}} + \frac{\partial \Psi}{\partial II_C} \frac{\partial II_C}{\partial \mathbf{C}} + \frac{\partial \Psi}{\partial III_C} \frac{\partial III_C}{\partial \mathbf{C}} \right)$$

where

$$\frac{\partial I_C}{\partial \mathbf{C}} = \mathbf{I}, \quad \frac{\partial II_C}{\partial \mathbf{C}} = 2\mathbf{C}, \quad \text{and} \quad \frac{\partial III_C}{\partial \mathbf{C}} = \det(\mathbf{C})\mathbf{C}^{-T} = j^2\mathbf{C}^{-1}$$

Assignment

- Discuss why the invariants on Pages 8 and 9 are invariant (ie. independent of rotation).
- Derive the formulas on page 11 from the formulas given in terms of \mathbf{C} on Page 8-9. If you have time derive an eigenvalue version of the invariant II_C^* .
- On page 38 we exploit the symmetry of the Cauchy stress tensor. Now prove that if $\sigma = \sigma^T$ then $\sigma : \mathbf{V}^T = \sigma : \mathbf{V}$ and that $\sigma : \mathbf{V} = \sigma : \mathbf{D}$.
- Derive the formula for \mathbf{S} on the page 46.
- Derive the stress-strain relation on the page 47 by differentiation of Ψ .

Assignment

Download <https://github.com/erleben/hyper-sim> run a bending rod simulation case and find a way to visualize the stress tensor field.

- Use the FEM method in the framework with the adaptive time stepping parameter on.
- Try and play around with the Young modulus and Poisson ratio values.

If you have time try to change the stress-strain relation used in the FVM or FEM methods to that of a neoHookean material (See Bonet and Wood text for examples on constitutive laws).