

Strain and Stress Tensors

Introduction to Concept and Definitions

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Modeling of Elastic Solids

The partial differential equation, Cauchy Equation, gives the equation of motion of an elastic solid

$$\rho \ddot{\mathbf{x}} = \mathbf{b} + \nabla \cdot \boldsymbol{\sigma}$$

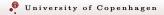
The boundary conditions of the Cauchy stress tensor σ are given by

 $\sigma \boldsymbol{n} = \boldsymbol{t}$

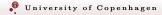
where t is a known applied surface traction. Finally, a constitutive law is needed that provides a relationship between stress and strain

$$\mathbf{S} = \frac{\partial \Psi}{\partial \mathbf{E}}$$

where **S** is 2. Piola–Kirchhoff stress tensor and **E** is the Green strain tensor. Now we will derive all these equations that we need.



Kinematics – Learning The Definitions of The Strain Tensors



The Material and Spatial Coordinates

We have two sets of coordinates

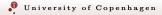
- Let **X** be undeformed coordinates
- and **x** the deformed coordinates

The deformation is given by the mapping from undeformed coordinates into deformed coordinates,

$$\boldsymbol{x} = \boldsymbol{\Phi}(\boldsymbol{X}, t)$$

Observe

- The same global reference coordinate system is used for both sets of coordinates.
- Undeformed coordinates are sometimes also called material coordinates and deformed coordinates are the spatial coordinates.



The Deformation Gradient

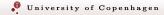
By definition of differentials we have

$$d\boldsymbol{x} = \underbrace{\frac{\partial \boldsymbol{\Phi}}{\partial \boldsymbol{X}}}_{\boldsymbol{\mathsf{F}}} d\boldsymbol{X}$$

The partial derivative $\frac{\partial \Phi}{\partial X} = \mathbf{F}$ is referred to as the Deformation gradient.

In particular, we have

$$\mathbf{F}_{ij} = \frac{\partial \mathbf{\Phi}_i}{\partial \mathbf{X}_j} = \frac{\partial \mathbf{x}_i}{\partial \mathbf{X}_j}$$



Investigating Local Deformation Measures

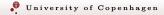
Think of dX_1 and dX_2 as arbitary chosen small "needles". By definition we have

 $d\mathbf{x}_1 = \mathbf{F} d\mathbf{X}_1$ $d\mathbf{x}_2 = \mathbf{F} d\mathbf{X}_2$

To describe any local deformation we investigate the dot products

 $d \mathbf{x}_1 \cdot d \mathbf{x}_2$ and $d \mathbf{X}_1 \cdot d \mathbf{X}_2$

as these hold information about any local angle or length deformations.



The Right Cauchy–Green Deformation Tensor

First we will try to express $dx_1 \cdot dx_2$ in terms of material coordinates,

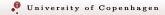
$$d\mathbf{x}_1 \cdot d\mathbf{x}_2 = d\mathbf{x}_1^T d\mathbf{x}_2$$

= $(\mathbf{F}d\mathbf{X}_1)^T (\mathbf{F}d\mathbf{X}_2)$
= $d\mathbf{X}_1^T \underbrace{(\mathbf{F}^T\mathbf{F})}_{\mathbf{C}} d\mathbf{X}_2$

where

 $\mathbf{C} = \mathbf{F}^{\mathcal{T}} \mathbf{F}$

is called the right Cauchy–Green Deformation tensor.



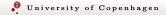
Tensor Invariants (1/2)

The eigenvalues of \mathbf{C} are independent of rotation thus the coefficients of the characteristic polynomial are invariants

$$\det \left(\mathbf{C} - \mathbf{I}\lambda\right) = -\lambda^3 + \mathbf{I}_{\mathcal{C}}\lambda^2 - \mathbf{II}_{\mathcal{C}}^*\lambda + \mathbf{III}_{\mathcal{C}} = 0$$

where

$$\begin{split} \mathbf{I}_{\mathcal{C}} &= \mathsf{tr}\left(\mathbf{C}\right) = \mathbf{C} : \mathbf{I} \\ \mathbf{II}_{\mathcal{C}}^{*} &= \frac{1}{2} \left(\mathsf{tr}\left(\mathbf{C}\right)^{2} - \mathsf{tr}\left(\mathbf{C}^{2}\right) \right) \\ \mathbf{III}_{\mathcal{C}} &= \mathsf{det}\left(\mathbf{C}\right) \end{split}$$

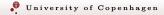


Tensor Invariants (2/2)

We prefer an alternative definition of the second invariant

$$\mathsf{II}_{\mathcal{C}} = \mathsf{tr}\left(\mathbf{C}^2
ight) = \mathbf{C}:\mathbf{C}$$

as it simplifies later equations



The Eigenvalue Decomposition

Since **C** is a symmetric positive definite we know that an eigenvalue decomposition exists

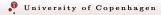
$$\mathbf{CN} - \mathbf{N} = 0$$

where

$$\mathbf{N} = \begin{bmatrix} \mathbf{N}_1 & \mathbf{N}_2 & \mathbf{N}_3 \end{bmatrix}$$
$$= \begin{bmatrix} \lambda_1^2 & 0 & 0\\ 0 & \lambda_2^2 & 0\\ 0 & 0 & \lambda_3^2 \end{bmatrix}$$

and $\mathbf{N}^T \mathbf{N} = \mathbf{N} \mathbf{N}^T = \mathbf{I}$ and $0 < \lambda_3^2 \le \lambda_2^2 \le \lambda_1^2 \in \mathbb{R}$. This can be rewritten as

$$\mathbf{C} = \mathbf{N}\mathbf{N}^{T} = \sum \lambda_{i}^{2} \left(\mathbf{N}_{i} \mathbf{N}_{i}^{T} \right)$$



Invariants in terms of Eigenvalues

Observe that knowing the eigenvalues λ_i^2 of **C** we can compute the tensor invariants as

$$I_{\mathcal{C}} = \lambda_1^2 + \lambda_2^2 + \lambda_3^2$$
$$II_{\mathcal{C}} = \lambda_1^4 + \lambda_2^4 + \lambda_3^4$$
$$II_{\mathcal{C}} = \lambda_1^2 \lambda_2^2 \lambda_3^2$$

These formulas will become handy when we later wish to change coordinates.

A Rotation Tensor

The square root of C is

 $\mathbf{C} = \sqrt{\mathbf{C}}\sqrt{\mathbf{C}} = \mathbf{U}\mathbf{U}$

with

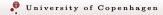
$$\mathbf{U} = \sum_{i} \lambda_{i} \left(\mathbf{N}_{i} \mathbf{N}_{i}^{T} \right)$$

Let us define the tensor ${\boldsymbol{\mathsf{R}}}$ as

$$\mathbf{R} = \mathbf{F}\mathbf{U}^{-1}$$

We can then prove that $\boldsymbol{\mathsf{R}}$ is a rotation tensor

$$\mathbf{R}^{T}\mathbf{R} = \mathbf{U}^{-T}\mathbf{F}^{T}\mathbf{F}\mathbf{U}^{-1} = \mathbf{U}^{-T}\mathbf{C}\mathbf{U}^{-1} = \mathbf{U}^{-T}\mathbf{U}\mathbf{U}\mathbf{U}^{-1} = \mathbf{I}$$



The Polar Decomposition of Deformation Gradient

The new rotation tensor \mathbf{R} implies that \mathbf{F} can be decomposed as

 $\mathbf{F} = \mathbf{R}\mathbf{U}$

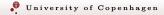
The implication is

$$d\boldsymbol{x} = \boldsymbol{\mathsf{F}}\boldsymbol{d}X = \boldsymbol{\mathsf{R}}\left(\boldsymbol{\mathsf{U}}\boldsymbol{d}\boldsymbol{X}\right)$$

where $(\mathbf{U}d\mathbf{X})$ is a stretch along the eigenvectors and **R** is a rotation of the stretched vector.

Therefore

- **U** is called the stretch tensor
- **R** is called the rotation tensor



The Left Cauchy–Green Deformation Tensor

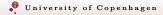
Next we will rewrite $dX_1 \cdot dX_2$ in terms of spatial coordinates,

$$d\boldsymbol{X}_{1} \cdot d\boldsymbol{X}_{2} = d\boldsymbol{X}_{1}^{T} d\boldsymbol{X}_{2}$$
$$= \left(\boldsymbol{\mathsf{F}}^{-1} d\boldsymbol{x}_{1}\right)^{T} \left(\boldsymbol{\mathsf{F}}^{-1} d\boldsymbol{x}_{2}\right)$$
$$= d\boldsymbol{x}_{1}^{T} \left(\underbrace{\boldsymbol{\mathsf{FF}}}_{\boldsymbol{\mathsf{B}}}^{T}\right)^{-1} d\boldsymbol{x}_{2}$$

where

 $\bm{\mathsf{B}} = \bm{\mathsf{F}}\bm{\mathsf{F}}^{\mathcal{T}}$

is called the left Cauchy-Green Deformation tensor.



The Lagrange–Green Strain Tensor

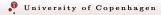
We look at the deformation change. The difference between spatial and material dot products expressed in material coordinates,

$$\frac{1}{2} (d\mathbf{x}_1 \cdot d\mathbf{x}_2 - d\mathbf{X}_1 \cdot d\mathbf{X}_2) = \frac{1}{2} (d\mathbf{x}_1^T d\mathbf{x}_2 - d\mathbf{X}_1^T d\mathbf{X}_2)$$
$$= \frac{1}{2} ((\mathbf{F}d\mathbf{X}_1)^T (\mathbf{F}d\mathbf{X}_2) - d\mathbf{X}_1^T d\mathbf{X}_2)$$
$$= d\mathbf{X}_1^T \underbrace{\frac{1}{2} (\mathbf{F}^T \mathbf{F} - \mathbf{I})}_{\mathbf{E}} d\mathbf{X}_2$$

where

$$\mathbf{E} = \frac{1}{2} \left(\mathbf{F}^{\mathsf{T}} \mathbf{F} - \mathbf{I} \right) = \frac{1}{2} \left(\mathbf{C} - \mathbf{I} \right)$$

is called the Green Strain tensor.



The Euler–Almansi Strain Tensor

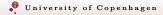
We describe the deformation change in terms of spatial coordinates,

$$\begin{aligned} \frac{1}{2} \left(d\mathbf{x}_1 \cdot d\mathbf{x}_2 - d\mathbf{X}_1 \cdot d\mathbf{X}_2 \right) &= \frac{1}{2} \left(d\mathbf{x}_1^T d\mathbf{x}_2 - d\mathbf{X}_1^T d\mathbf{X}_2 \right) \\ &= \frac{1}{2} \left(d\mathbf{x}_1^T d\mathbf{x}_2 - \left(\mathbf{F}^{-1} d\mathbf{x}_1 \right)^T \left(\mathbf{F}^{-1} d\mathbf{x}_2 \right) \right) \\ &= d\mathbf{x}_1^T \underbrace{\frac{1}{2} \left(\mathbf{I} - \left(\mathbf{F} \mathbf{F}^T \right)^{-1} \right)}_{\mathbf{g}} d\mathbf{x}_2 \end{aligned}$$

where

$$\mathbf{e} = \frac{1}{2} \left(\mathbf{I} - \left(\mathbf{F} \mathbf{F}^{\mathcal{T}} \right)^{-1} \right) = \frac{1}{2} \left(\mathbf{I} - \mathbf{B}^{-1} \right)$$

is called the Euler Strain tensor.



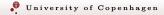
Spatial to Material Tensor Conversions

Observe that

$$\mathbf{e} = \mathbf{F}^{-T} \mathbf{E} \mathbf{F}^{-1}$$

 $\mathbf{E} = \mathbf{F}^{T} \mathbf{e} \mathbf{F}$

These are general tensor conversion formulas.



The Displacement Field

Let us define the displacement field as follows

$$\boldsymbol{u} = \boldsymbol{x} - \boldsymbol{X}$$

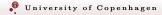
Then we trivially have

 $\boldsymbol{x} = \boldsymbol{u} + \boldsymbol{X}$

and

$$\mathbf{F} = \frac{\partial (\boldsymbol{u} + \boldsymbol{X})}{\partial \boldsymbol{X}} = \frac{\partial \boldsymbol{u}}{\partial \boldsymbol{X}} + \mathbf{I}$$
$$\frac{\partial \boldsymbol{u}}{\partial \boldsymbol{X}} = \mathbf{F} - \mathbf{I}$$

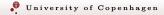
or



Putting Displacement Field into Play

Let us use $\mathbf{F} = \frac{\partial \mathbf{u}}{\partial \mathbf{X}} + \mathbf{I}$ in the definition of the Green Strain Tensor

$$\mathbf{E} = \frac{1}{2} \left(\left(\frac{\partial \boldsymbol{u}}{\partial \boldsymbol{X}} + \mathbf{I} \right)^{T} \left(\frac{\partial \boldsymbol{u}}{\partial \boldsymbol{X}} + \mathbf{I} \right) - \mathbf{I} \right)$$
$$= \frac{1}{2} \left(\left(\frac{\partial \boldsymbol{u}}{\partial \boldsymbol{X}}^{T} \frac{\partial \boldsymbol{u}}{\partial \boldsymbol{X}} \right) + \left(\mathbf{I}^{T} \frac{\partial \boldsymbol{u}}{\partial \boldsymbol{X}} \right) + \left(\frac{\partial \boldsymbol{u}}{\partial \boldsymbol{X}}^{T} \mathbf{I} \right) + \left(\mathbf{I}^{T} \mathbf{I} \right) - \mathbf{I} \right)$$
$$= \frac{1}{2} \left(\frac{\partial \boldsymbol{u}}{\partial \boldsymbol{X}}^{T} \frac{\partial \boldsymbol{u}}{\partial \boldsymbol{X}} + \frac{\partial \boldsymbol{u}}{\partial \boldsymbol{X}} + \frac{\partial \boldsymbol{u}}{\partial \boldsymbol{X}}^{T} \right)$$



Assuming Small Displacement Gradients

If
$$\| \frac{\partial u}{\partial X} \| \ll 1$$
 then $\frac{\partial u}{\partial X}^T \frac{\partial u}{\partial X} \approx \mathbf{0}$ and we have

$$\mathbf{E} \approx \varepsilon_0 = \frac{1}{2} \left(\frac{\partial \boldsymbol{u}}{\partial \boldsymbol{X}}^{\mathsf{T}} + \frac{\partial \boldsymbol{u}}{\partial \boldsymbol{X}} \right)$$

This strain tensor is called the Cauchy Strain tensor. If we used ${\bf e}$ instead of ${\bf E}$ we would have found

$$\mathbf{e} \approx \varepsilon = \frac{1}{2} \left(\frac{\partial \mathbf{u}}{\partial \mathbf{x}}^{T} + \frac{\partial \mathbf{u}}{\partial \mathbf{x}} \right)$$

However, for small displacement gradients $x \approx X$, $\frac{\partial u}{\partial x} \approx \frac{\partial u}{\partial X}$, and so $\varepsilon_0 \approx \varepsilon$.

The Strain Tensors

Green Strain Tensor

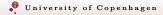
$$\mathbf{E} = \frac{1}{2} \left(\frac{\partial \boldsymbol{u}}{\partial \boldsymbol{X}}^{\mathsf{T}} \frac{\partial \boldsymbol{u}}{\partial \boldsymbol{X}} + \frac{\partial \boldsymbol{u}}{\partial \boldsymbol{X}} + \frac{\partial \boldsymbol{u}}{\partial \boldsymbol{X}}^{\mathsf{T}} \right)$$

Cauchy Strain Tensor

$$\varepsilon \approx \varepsilon_0 = \frac{1}{2} \left(\frac{\partial \boldsymbol{u}}{\partial \boldsymbol{X}}^T + \frac{\partial \boldsymbol{u}}{\partial \boldsymbol{X}} \right)$$

Observe that using $\frac{\partial u}{\partial X} = \mathbf{F} - \mathbf{I}$ we have

$$\varepsilon_0 = \frac{1}{2} \left(\mathbf{F}^T + \mathbf{F} \right) - \mathbf{I}$$



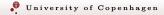
The Velocity

Since $\mathbf{x} = \Phi(\mathbf{X}, t)$ then the spatial velocity of a material point is

$$\mathbf{v}(\mathbf{X},t) \equiv \dot{\mathbf{x}}(\mathbf{X},t) = \frac{\partial \mathbf{x}(\mathbf{X},t)}{\partial t} = \frac{\partial \Phi\left(\mathbf{X},t\right)}{\partial t}$$

or given in terms of spatial coordinates

$$oldsymbol{v}(oldsymbol{x},t)=oldsymbol{v}\left(\Phi^{-1}\left(oldsymbol{x},t
ight),t
ight)$$



The Velocity Gradient

The velocity gradient wrt. spatial coordinates is then

$$\mathbf{V} \equiv \frac{\partial \mathbf{v}(\mathbf{x}, t)}{\partial \mathbf{x}}$$

Observe (Notice $\frac{\partial \mathbf{X}}{\partial t} = 0$) $\dot{\mathbf{F}} = \frac{d}{dt} \left(\frac{\partial \Phi}{\partial \mathbf{X}} \right) = \frac{\partial}{\partial \mathbf{X}} \left(\frac{\partial \Phi}{\partial t} \right) = \frac{\partial}{\partial \mathbf{X}} \mathbf{v} = \frac{\partial \mathbf{v}}{\partial \mathbf{x}} \frac{\partial \Phi}{\partial \mathbf{X}} = \mathbf{VF}$

Resulting in

$$\mathbf{V} = \dot{\mathbf{F}}\mathbf{F}^{-1}$$



The Rate of Deformation

Taking the derivative

$$\frac{d}{dt}(d\boldsymbol{x}_1 \cdot d\boldsymbol{x}_2) = d\boldsymbol{X}_1^T \dot{\boldsymbol{\mathsf{C}}} d\boldsymbol{X}_2 = 2d\boldsymbol{X}_1^T \dot{\boldsymbol{\mathsf{E}}} d\boldsymbol{X}_2$$

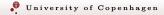
because $\mathbf{E} = \frac{1}{2}(\mathbf{C} - \mathbf{I})$. The material strain rate tensor is

$$\dot{\mathbf{E}} = rac{1}{2}\dot{\mathbf{C}} = rac{1}{2}\left(\dot{\mathbf{F}}^{T}\mathbf{F} + \mathbf{F}^{T}\dot{\mathbf{F}}
ight)$$

Using $d\boldsymbol{X} = \boldsymbol{\mathsf{F}}^{-1} d\boldsymbol{x}$ we have

$$\frac{1}{2}\frac{d}{dt}\left(d\mathbf{x}_{1}\cdot d\mathbf{x}_{2}\right) = d\mathbf{x}_{1}^{T}\underbrace{\mathbf{F}^{-T}\dot{\mathbf{E}}\mathbf{F}^{-1}}_{\mathbf{D}}d\mathbf{x}_{2}$$

where **D** is the rate of deformation tensor



Coordinate Conversion of Rate Tensors

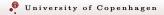
Observe that similar to the Euler-Almansi and Lagrange-Green strain tensors we have

 $\mathbf{D} = \mathbf{F}^{-T} \dot{\mathbf{E}} \mathbf{F}^{-1}$ $\dot{\mathbf{E}} = \mathbf{F}^{T} \mathbf{D} \mathbf{F}$

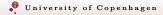
From this, we also find that

$$\begin{split} \mathbf{D} &= \mathbf{F}^{-T} \dot{\mathbf{E}} \mathbf{F}^{-1} \\ &= \mathbf{F}^{-T} \frac{1}{2} \left(\dot{\mathbf{F}}^{T} \mathbf{F} + \mathbf{F}^{T} \dot{\mathbf{F}} \right) \mathbf{F}^{-1} \\ &= \frac{1}{2} \left(\mathbf{V} + \mathbf{V}^{T} \right) \end{split}$$

because $\mathbf{V} = \dot{\mathbf{F}}\mathbf{F}^{-1}$. Thus **D** is the symmetric part of the velocity gradient.



Dynamics – Learning The Definitions of The Stress Tensors and Power



A Spatial Surface Element

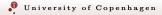
The spatial surface element ds can be written as

$$dm{s} = egin{bmatrix} dm{s}_1 \ dm{s}_2 \ dm{s}_3 \end{bmatrix}$$

where ds_i is the projection of the area of surface ds onto the *i*th coordinate axis.

Observe that the surface normal is given by

$$\boldsymbol{n} = rac{d\boldsymbol{s}}{\parallel d\boldsymbol{s} \parallel}$$



Cauchy's Stress Hypothesis

Definition: Stress is force per unit area. If we apply the force df to the surface element ds

$$d\boldsymbol{f}_1 = \sigma_{11}d\boldsymbol{s}_1 + \sigma_{12}d\boldsymbol{s}_3 + \sigma_{13}d\boldsymbol{s}_3$$
$$d\boldsymbol{f}_2 = \sigma_{21}d\boldsymbol{s}_1 + \sigma_{22}d\boldsymbol{s}_3 + \sigma_{23}d\boldsymbol{s}_3$$
$$d\boldsymbol{f}_3 = \sigma_{31}d\boldsymbol{s}_1 + \sigma_{32}d\boldsymbol{s}_3 + \sigma_{33}d\boldsymbol{s}_3$$

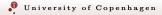
where σ_{ij} depends on position and time, collecting them

$$\sigma = \left[\sigma_{ij}\right]$$

we have

$$d\mathbf{f} = \sigma d\mathbf{s}$$

where σ is the Cauchy Stress Tensor.



The First Piola-Kirchhoff Stress Tensor

By definition the Cauchy stress tensor

$$d\mathbf{f} = \sigma d\mathbf{s}$$

The first Piola-Kirchhoff stess tensor is defined as

 $d\boldsymbol{f} = \mathbf{P}d\boldsymbol{S}$

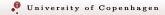
So we must have

$$\sigma ds = df = \mathbf{P} dS$$

Using Nanson's Relation

$$\sigma j \mathbf{F}^{-T} d\mathbf{S} = d\mathbf{f} = \mathbf{P} d\mathbf{S}$$

From this we have the relation between the Cauchy stress tensor and the first Piola–Kirchhoff stress tensor



The Second Piola-Kirchhoff Stress Tensor

By definition the Cauchy stress tensor

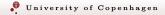
$$d\mathbf{f} = \sigma d\mathbf{s}$$

The second Piola-Kirchhoff stess tensor is defined as

 $d\textbf{\textit{F}}=\textbf{S}d\textbf{\textit{S}}$

However $d\mathbf{f} = \mathbf{F} d\mathbf{F}$ so we must have

$$\sigma ds = df = FdF = FSdS$$
$$\sigma ds = FSdS$$
$$jF^{-1}\sigma F^{-T}dS = SdS$$



The Traction Force

We have

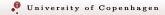
$$d\mathbf{f} = \sigma d\mathbf{s}$$

If we divide by surface area we will have traction $m{t} = dm{f} / \parallel dm{s} \parallel$

 $\boldsymbol{t} = \sigma \boldsymbol{n}$

where

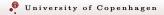
$$\begin{bmatrix} \mathbf{t}_1 \\ \mathbf{t}_2 \\ \mathbf{t}_3 \end{bmatrix} = \begin{bmatrix} \sigma_{11} & \sigma_{12} & \sigma_{13} \\ \sigma_{21} & \sigma_{22} & \sigma_{23} \\ \sigma_{31} & \sigma_{32} & \sigma_{33} \end{bmatrix} \begin{bmatrix} \mathbf{n}_1 \\ \mathbf{n}_2 \\ \mathbf{n}_3 \end{bmatrix}$$



Integration of Forces

Given the body volume force density \boldsymbol{b} and arbitary spatial volume element v with spatial surface s then the total force on the volume is found by intergration

$$\mathcal{F} = \int_{\mathbf{v}} \mathbf{b} d\mathbf{v} + \oint_{\mathbf{s}} \mathbf{t} d\mathbf{s}$$
$$= \int_{\mathbf{v}} \mathbf{b} d\mathbf{v} + \oint_{\mathbf{s}} (\sigma \mathbf{n}) d\mathbf{s}$$



Using Volume Integrals

Recall Gauss–Divergence Theorem, for tensor field **A** over any closed volume v with surface s

$$\int_{V}
abla \cdot \mathbf{A} dv = \oint_{s} \mathbf{A} n ds$$

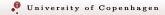
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Applying this to

$$\mathcal{F} = \int_{v} \boldsymbol{b} dv + \oint_{s} \sigma \boldsymbol{n} ds$$

yields

$$\boldsymbol{\mathcal{F}} = \int_{\boldsymbol{v}} \underbrace{(\boldsymbol{b} + \nabla \cdot \boldsymbol{\sigma})}_{\boldsymbol{f}^*} d\boldsymbol{v}$$



The Total Force and the Effective Force

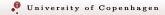
In mechanical equilibrium, the effective force f^* must be zero since v was chosen arbitrarily. Thus writing it out

$$\boldsymbol{b} + \nabla \cdot \boldsymbol{\sigma} = 0$$

This is Cauchy's Equation of Equilibrium. In non equilibrium, we must take inertia forces into account, $\mathcal{F} = \int_{v} \rho \ddot{x} dv$,

$$\rho \ddot{\mathbf{x}} = \mathbf{b} + \nabla \cdot \sigma$$

This is the motion of the equation in spatial coordinates for any continuum.



Recap Work and Power Definitions

Let $\mathbf{x}(t)$ be a particle and \mathbf{f} some force acting on the particle. Work is force times distance, for a constant force over a displacement \mathbf{u}

$$W = \boldsymbol{f} \cdot \boldsymbol{u}$$

Or more general

$$W = \int \boldsymbol{f} \cdot d\boldsymbol{x}$$

Power is the rate at which work is performed

$$P \equiv rac{dW}{dt}$$

The instantaneous power is $P = \mathbf{f} \cdot \mathbf{v}$ and so $W = \int_0^t \mathbf{f} \cdot \mathbf{v} dt$.

Power of The Effective Force

The power from the effective force

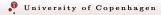
$$p = f^* \cdot v$$

The total power applied by the effective force

$$P = \int_{v} p dv = \int_{v} \boldsymbol{f}^* \cdot \boldsymbol{v} dv$$

Rewriting into

$$P = \int_{v} (\boldsymbol{b} + (\nabla \cdot \sigma)) \cdot \boldsymbol{v} dv$$
$$= \int_{v} (\nabla \cdot \sigma) \cdot \boldsymbol{v} dv + \int_{v} \boldsymbol{b} \cdot \boldsymbol{v} dv$$



Using the "Product" Rule

Using

$$\nabla \cdot (\sigma \mathbf{v}) = (\nabla \cdot \sigma) \cdot \mathbf{v} + \sigma : (\nabla \mathbf{v}^{\mathsf{T}})$$

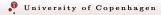
Then

$$P = \int_{\boldsymbol{v}} (\nabla \cdot \boldsymbol{\sigma}) \cdot \boldsymbol{v} d\boldsymbol{v} + \int_{\boldsymbol{v}} \boldsymbol{b} \cdot \boldsymbol{v} d\boldsymbol{v}$$

Becomes

$$P = \int_{V} \nabla \cdot (\sigma \mathbf{v}) d\mathbf{v} - \int_{V} \sigma : \mathbf{V}^{\mathsf{T}} d\mathbf{v} + \int_{V} \mathbf{b} \cdot \mathbf{v} d\mathbf{v}$$

Recall $\nabla \boldsymbol{v}^T = \boldsymbol{V}^T$.



Apply Gauss–Divergence Theorem

Using Gauss–Divergence Theorem

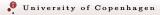
$$P = \oint_{s} \boldsymbol{n} \cdot (\sigma \boldsymbol{v}) ds - \int_{v} \sigma : \boldsymbol{V}^{T} dv + \int_{v} \boldsymbol{b} \cdot \boldsymbol{v} dv$$

Using $\boldsymbol{t} = \sigma \boldsymbol{n}$ and $\sigma^T = \sigma$

$$P = \oint_{s} oldsymbol{t} \cdot oldsymbol{v} ds - \int_{v} \sigma : oldsymbol{V} dv + \int_{v} oldsymbol{b} \cdot oldsymbol{v} dv$$

Using $\mathbf{D} = \frac{1}{2} \left(\mathbf{V} + \mathbf{V}^T \right)$ and symmetry of σ

$$P = \oint_{s} \boldsymbol{t} \cdot \boldsymbol{v} ds - \int_{v} \sigma : \boldsymbol{\mathsf{D}} dv + \int_{v} \boldsymbol{b} \cdot \boldsymbol{v} dv$$



The Power in Spatial Coordinates

The internal stress power term in spatial coordinates

$$P_{\mathsf{e}} = \int_{\mathsf{v}} \sigma : \mathbf{V} d\mathsf{v} = \int_{\mathsf{v}} \sigma : \mathbf{D} d\mathsf{v}$$

Thus, in terms of power, we say that σ is work conjugate to **D**. Our next task is to rewrite the internal power in terms of material coordinates. The result is to find the work conjugate quantities of

- The First Piola-Kirchhoff Stress tensor
- The Second Piola-Kirchhoff Stress tensor

Power Conjugency of ${\boldsymbol{\mathsf{P}}}$

$$P_{e} = \int_{V} \sigma : \mathbf{V} dv$$

= $\int_{V} j\sigma : \mathbf{V} dV$
= $\int_{V} j\sigma : (\dot{\mathbf{F}}\mathbf{F}^{-1}) dV$
= $\int_{V} \operatorname{tr} (j\sigma (\dot{\mathbf{F}}\mathbf{F}^{-1})) dV$
= $\int_{V} \operatorname{tr} ((j\mathbf{F}^{-1}\sigma) \dot{\mathbf{F}}) dV$
= $\int_{V} \mathbf{P} : \dot{\mathbf{F}} dV$

(by definition) (by dv = jdV) (by $\mathbf{V} = \dot{\mathbf{F}}\mathbf{F}^{-1}$) (by $\mathbf{A} : \mathbf{B} = \operatorname{tr}(\mathbf{A}^T \mathbf{B})$) (tr(AB) = tr(BA))(by tr $(\mathbf{A}^T \mathbf{B}) = \mathbf{A} : \mathbf{B}$)

Power Conjugency of $\boldsymbol{\mathsf{S}}$

$$P_{e} = \int_{V} \sigma : \mathbf{D} dv$$

= $\int_{V} j\sigma : \mathbf{D} dV$
= $\int_{V} j\sigma : (\mathbf{F}^{-T} \dot{\mathbf{E}} \mathbf{F}^{-1}) dV$
= $\int_{V} \operatorname{tr} \left(j\sigma \left(\mathbf{F}^{-T} \dot{\mathbf{E}} \mathbf{F}^{-1} \right) \right) dV$
= $\int_{V} \operatorname{tr} \left(\left(j\mathbf{F}^{-1}\sigma\mathbf{F}^{-T} \right) \dot{\mathbf{E}} \right) dV$
= $\int_{V} \mathbf{S} : \dot{\mathbf{E}} dV$

(by definition) (by dv = jdV) (by $\mathbf{D} = \mathbf{F}^{-T} \dot{\mathbf{E}} \mathbf{F}^{-1}$) (by $\mathbf{A} : \mathbf{B} = \operatorname{tr}(\mathbf{A}^T \mathbf{B})$) (by tr $(\mathbf{AB}) = tr (\mathbf{BA})$) (by tr $(\mathbf{A}^T \mathbf{B}) = \mathbf{A} : \mathbf{B}$)



The Energy Balance

For the equilibrium system mechanical energy balance means that P = 0. That is the time derivative of mechanical energy is zero – constant in time.

In the case of a non-equilibrium system we must include inertia forces,

$$P = \int_{V} (\rho \ddot{\mathbf{x}} - \mathbf{b} - \nabla \cdot \sigma) \cdot \mathbf{v} dv$$

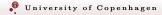
Repeating the previous derivations to the total power becomes

$$P = \int_{v} \rho \ddot{\boldsymbol{x}} \cdot v dv + \int_{v} \sigma : \boldsymbol{\mathsf{D}} dv - \oint_{s} \boldsymbol{t} \cdot \boldsymbol{v} dv - \int_{v} \boldsymbol{b} \cdot \boldsymbol{v} dv$$

and it must be zero for the conservation of mechanical energy.



Constitutive Equations



The Strain Energy in terms of **P**

The total strain energy Ψ (The time integral of the "elastic" power $P_{\rm e}$),

$$\Psi(\mathbf{F}) = \int_0^t \underbrace{\mathbf{P} : \dot{\mathbf{F}}}_{=\dot{\Psi}} dt$$

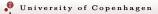
By the chain rule, we have

$$\dot{\Psi} = \frac{d}{dt}\Psi(\mathbf{F}) = \frac{\partial\Psi}{\partial\mathbf{F}} : \dot{\mathbf{F}}$$
$$\mathbf{P} : \dot{\mathbf{F}} = \dot{\Psi} = \frac{\partial\Psi}{\partial\mathbf{F}} : \dot{\mathbf{F}}$$

and we must have

So

$$\mathbf{P} = \frac{\partial \Psi}{\partial \mathbf{F}}$$



The Strain Energy in terms of \boldsymbol{S}

The total strain energy Ψ ,

$$\Psi(\mathbf{F}) = \int_0^t \underbrace{\mathbf{S}: \dot{\mathbf{E}}}_{=\dot{\Psi}} dt$$

By the chain rule, we have

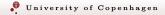
$$\dot{\Psi} = rac{d}{dt}\Psi(\mathbf{E}) = rac{\partial\Psi}{\partial\mathbf{E}}:\dot{\mathbf{E}}$$

$$\mathbf{S}:\dot{\mathbf{E}}=\dot{\Psi}=rac{\partial\Psi}{\partial\mathbf{E}}:\dot{\mathbf{E}}$$

and we must have

So

$$\mathbf{S} = \frac{\partial \Psi}{\partial \mathbf{E}}$$



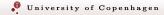
Isotropic Materials

The potential function is a function of invariants of ${f C}$

$$\Psi(\mathbf{C}) = \Psi(\mathbf{I}_{\mathcal{C}}, \mathbf{II}_{\mathcal{C}}, \mathbf{III}_{\mathcal{C}})$$

The Second Piola-Kirchhoff Stress Tensor computed as

$$\mathbf{S} = 2\frac{\partial\Psi}{\partial I_{C}}\mathbf{I} + 4\frac{\partial\Psi}{\partial II_{C}}\mathbf{C} + 2j^{2}\frac{\partial\Psi}{\partial III_{C}}\mathbf{C}^{-1}$$



The Saint Venant-Kirchhoff (SVK) Material Model

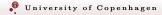
The simplest hyperelastic material model. The strain-energy is

$$\Psi(\mathbf{E}) = rac{\lambda}{2} \left(\operatorname{tr}\left(\mathbf{E}
ight)
ight)^2 + \mu \operatorname{tr}\left(\mathbf{E}^2
ight)$$

where λ and μ are the Lamé constants.

From ${f S}=rac{\partial\Psi}{\partial{f E}}$ we have the stress-strain relation

$$\mathbf{S} = \lambda \mathrm{tr}\left(\mathbf{E}\right)\mathbf{I} + 2\mu\mathbf{E}$$



The S-C Relation using the SVK-Material

By definition $2\mathbf{E} + \mathbf{I} = \mathbf{C}$

$$\mathbf{S} = rac{\partial \Psi(\mathbf{C})}{\partial \mathbf{C}} rac{\partial \mathbf{C}}{\partial \mathbf{E}} = 2 rac{\partial \Psi}{\partial \mathbf{C}}$$

For isotropic materials stretch is independent of rotation $\Psi(\mathbf{C}) = \Psi(\mathbf{I}_{C}, \mathbf{II}_{C}, \mathbf{II}_{C})$

$$\mathbf{S} = 2\left(\frac{\partial\Psi}{\partial I_{C}}\frac{\partial I_{C}}{\partial \mathbf{C}} + \frac{\partial\Psi}{\partial II_{C}}\frac{\partial I_{C}}{\partial \mathbf{C}} + \frac{\partial\Psi}{\partial II_{C}}\frac{\partial II_{C}}{\partial \mathbf{C}}\right)$$

where

$$\frac{\partial I_{\mathcal{C}}}{\partial \mathbf{C}} = \mathbf{I}, \quad \frac{\partial II_{\mathcal{C}}}{\partial \mathbf{C}} = 2\mathbf{C}, \quad \text{and} \quad \frac{\partial III_{\mathcal{C}}}{\partial \mathbf{C}} = \det(\mathbf{C})\mathbf{C}^{-\mathcal{T}} = j^2\mathbf{C}^{-1}$$



Assignment

- Discuss why the invariants on Pages 8 and 9 are invariant (ie. independent of rotation).
- Derive the formulas on page 11 from the formulas given in terms of **C** on Page 8-9. If you have time derive an eigenvalue version of the invariant II^{*}_C.
- On page 38 we exploit the symmetry of the Cauchy stress tensor. Now prove that if $\sigma = \sigma^T$ then $\sigma : \mathbf{V}^T = \sigma : \mathbf{V}$ and that $\sigma : \mathbf{V} = \sigma : \mathbf{D}$.
- Derive the formula for **S** on the page 46.
- Derive the stress-strain relation on the page 47 by differentiation of Ψ .



Assignment

Download https://github.com/erleben/hyper-sim run a bending rod simulation case and find a way to visualize the stress tensor field.

- Use the FEM method in the framework with the adaptive time stepping parameter on.
- Try and play around with the Young modulus and Poisson ratio values.

If you have time try to change the stress-strain relation used in the FVM or FEM methods to that of a neoHookean material (See Bonet and Wood text for examples on constitutive laws).